# Towards an Optimal Substructuring Method for Model Reduction 

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#### Abstract

Substructuring methods have been studied since 1960s. The modes of subsystems associated with the lowest frequencies are typically retained. This mode selection rule is largely heuristic. In this paper, we use a moment-matching analysis tool to derive a new mode selection criterion, which is compatible to the one recently derived by Givoli et al using Dirichlet-to-Neumann (DtN) map as an analysis tool. The improvements of the new mode selection criterion are demonstrated by numerical examples from structural dynamics and MEMS simulation.


## 1 Introduction

Model-order reduction techniques play an indispensable role to meet the continual and compelling need for accurately and efficiently simulating dynamical behavior of large and complex physical systems. One popular method is the substructuring or the component mode synthesis (CMS), which was developed back to early 1960s [7]8|4]. CMS explicitly exploits underlying structures of a system and effectively avoids the expenses of processing the entire system at once. The model-order reduction of subsystems can be conducted in parallel. The subsystem structure is preserved.

Specifically, in this paper, we consider a lumped MIMO dynamical system of the form

$$
\Sigma_{N}:\left\{\begin{align*}
M \ddot{x}(t)+K x(t) & =B u(t),  \tag{1.1}\\
y(t) & =L^{T} x(t)
\end{align*}\right.
$$

with the initial conditions $x(0)=x_{0}$ and $\dot{x}(0)=v_{0}$. Here $t$ is the time variable, $x(t) \in \mathcal{R}^{N}$ is a state vector, $N$ is the degree of freedoms (DOFs), $u(t) \in \mathcal{R}^{p}$ the input excitation force vector, and $y(t) \in \mathcal{R}^{m}$ the output measurement vector. $B \in \mathcal{R}^{N \times p}$ and $L \in \mathcal{R}^{N \times m}$ are input and output distribution arrays, respectively. $M$ and $K$ are system matrices, such as mass and stiffness. Assume that $M$ is symmetric positive definite and $K$ is symmetric semidefinite. Furthermore, the state vector $x(t)$ and the system matrices $M$ and $K$ are posed of subsystem structure, namely, they are partitioned into the three blocks, representing subsystems I, II and interface:

$$
x(t)=\left[\begin{array}{l}
x_{1}(t)  \tag{1.2}\\
x_{2}(t) \\
x_{3}(t)
\end{array}\right], M=\left[\begin{array}{lll}
M_{11} & & M_{13} \\
& M_{22} & M_{23} \\
M_{13}^{T} & M_{23}^{T} & M_{33}
\end{array}\right], K=\left[\begin{array}{lll}
K_{11} & & K_{13} \\
& K_{22} & K_{23} \\
K_{13}^{T} & K_{23}^{T} & K_{33}
\end{array}\right] .
$$

We denote the number of DOFs of subsystems I, II and the interface by $N_{1}, N_{2}$ and $N_{3}$, respectively. Thus the total number of DOFs of $\Sigma_{N}$ is $N=N_{1}+N_{2}+N_{3}$.

By Laplace transform, the input-output behavior of $\Sigma_{N}$ in the frequency domain is characterized by the transfer function

$$
H(\omega)=L^{T}\left(-\omega^{2} M+K\right)^{-1} B
$$

where $\omega$ is referred to as the frequency. For the simplicity of exposition, we have assumed that $x(0)=\dot{x}(0)=0$.

A substructuring method replaces the system $\Sigma_{N}$ with a system of the same form but (much) smaller dimension of the state-vector $z(t)$ :

$$
\Sigma_{n}:\left\{\begin{align*}
M_{n} \ddot{z}(t)+K_{n} z(t) & =B_{n} \mathbf{u}(t),  \tag{1.3}\\
\widehat{y}(t) & =L_{n}^{T} z(t),
\end{align*}\right.
$$

such that the input-output behavior of $\Sigma_{n}$ is an acceptable approximation of $\Sigma_{N}$. The number of DOFs of the new state-vector $z(t)$ is $n=n_{1}+n_{2}+N_{3}$ with $n_{1}<N_{1}$ and $n_{2}<N_{2}$. The DOFs of the interface block is unchanged. Furthermore, $M_{n}$ and $K_{n}$ preserve the block structures of $M$ and $K$.

A key step in substructuring methods is to compute and retain the modes of subsystems. A standard mode selection practice is to retain the modes associated with few lowest frequencies. This is largely heuristic and does not guarantee to produce an optimal reduced system $\Sigma_{n}$ as shown by the following simple example. Let

$$
M=\left[\begin{array}{c|cc|c}
1 & & & 0.7  \tag{1.4}\\
\hline & \mathbf{1} & & 10^{-3} \\
& & \mathbf{1} & 0.3 \\
\hline 0.7 & 10^{-3} & 0.3 & 1
\end{array}\right], K=\left[\begin{array}{c|c|c}
0.9 & & \\
\hline & \mathbf{1} & \\
& \mathbf{2} & \\
\hline & & 1
\end{array}\right], B=\left[\begin{array}{c}
\frac{1}{0} \\
\frac{0}{0}
\end{array}\right], L=\left[\begin{array}{c}
\frac{1}{1} \\
\frac{1}{1}
\end{array}\right] .
$$

Suppose the subsystem II is reduced. Then by the standard lowest frequency mode selection criterion, the reduced system $\Sigma_{n}$ is given by

$$
M_{n}=\left[\begin{array}{c|c|c}
1 & & 0.7  \tag{1.5}\\
\hline & \mathbf{1} & 10^{-3} \\
\hline 0.7 & 10^{-3} & 1
\end{array}\right], K_{n}=\left[\begin{array}{l|l}
0.9 & \\
\hline & \mathbf{1} \\
\hline & \\
\hline
\end{array}\right], B_{n}=\left[\begin{array}{c}
\frac{1}{0} \\
\hline 0
\end{array}\right], L_{n}=\left[\begin{array}{c}
\frac{1}{1} \\
\hline 1
\end{array}\right] .
$$

However, if we retain the other mode in the system II, then the reduced system $\widehat{\Sigma}_{n}$ is given by

$$
\widehat{M}_{n}=\left[\begin{array}{c|c|c}
1 & & 0.7  \tag{1.6}\\
\hline & \mathbf{1} & 0.3 \\
\hline 0.7 & 0.3 & 1
\end{array}\right], \widehat{K}_{n}=\left[\begin{array}{l|l}
0.9 & \\
\hline & \mathbf{2} \\
\hline & \\
\hline & 1
\end{array}\right], \widehat{B}_{n}=\left[\begin{array}{c}
\frac{1}{0} \\
\frac{0}{0}
\end{array}\right], \widehat{L}_{n}=\left[\begin{array}{c}
\frac{1}{1} \\
\hline 1
\end{array}\right] .
$$

Figure 1 shows the magnitudes (in log of base 10) of the transfer function $H(\omega)$ of the original system $\Sigma_{N}$ and the reduced ones $H_{n}(\omega)$ (called CMS line) and $\widehat{H}_{n}(\omega)$ (called $\mathrm{CMS}_{\chi}$ line). It is clear that the low-frequency dominant mode selection criterion is not optimal.


Fig. 1. The frequency response analysis (top) and relative error (bottom) for the miniature example

A question that arises naturally is "which are the important modes of subsystems?" In the recent work of Givoli et al [16], an optimal modal reduction (OMR) algorithm is proposed. In contrast to the low-frequency dominant mode selection rule, they introduce the concept of coupling matrix-based mode selection criterion. The concept is derived via the DtN map analysis tool, originally developed for solving partial differential equations with non-reflecting boundary conditions [9]. They show that the OMR method is better than the standard modal reduction (SMR) method. However, there are a number of limitations in the OMR method, such as the assumption of external force $B u(t)$ only applied to one of the subsystems.

In this paper, we present an alternative mode selection criterion to the CMS method. The resulting method is called $\mathrm{CMS}_{\chi}$. The new mode selection criterion in $\mathrm{CMS}_{\chi}$ is derived in an algebraic setting based on the concept of moment-matching in frequency domain. It coincides with the coupling matrix-based mode selection criterion used in the OMR method. However, mathematical derivation of moment-matching based mode selection criterion is much simpler than the DtN mapping based derivation used in OMR. Moreover, it does not need the assumption of the special form of external force $B u(t)$ as used in OMR.

## 2 Substructuring Methods

In this section, we first discuss a generic CMS method, which is based on the original CMS developed by Hurty [718] and Craig and Bampton [4]. Then we specify the
difference between the standard CMS method and the new one we propose. We give a justification for the new method in the next section.

In a generic and compact form, the key step of the CMS method is on the construction of the transformation matrix $V_{n}$ of the form

$$
V_{n}=\begin{gather*}
N_{1}  \tag{2.7}\\
N_{2} \\
N_{3}
\end{gather*}\left(\begin{array}{ccc}
n_{1} & n_{2} & N_{3} \\
\Phi_{1} & & \Psi_{13} \\
& \Phi_{2} & \Psi_{23} \\
& & I_{N_{3}}
\end{array}\right),
$$

where $\Psi_{i 3}=-K_{i i}^{-1} K_{i 3}$ for $i=1,2$, and $\Phi_{i}$ is an $N_{i} \times n_{i}$ matrix whose columns are the selected $n_{i}$ eigenvectors $\phi_{j}^{(i)}$ of the matrix pair $\left(M_{i i}, K_{i i}\right)$ :

$$
\begin{equation*}
K_{i i} \phi_{j}^{(i)}=\lambda_{j}^{(i)} M_{i i} \phi_{j}^{(i)} \quad \text { and } \quad\left(\phi_{j}^{(i)}\right)^{T} M_{i i} \phi_{k}^{(i)}=\delta_{j k}, \tag{2.8}
\end{equation*}
$$

where $\delta_{j k}$ is the Kronecker delta. In structural dynamics, $\Phi_{i}$ is the interior partition of the fixed-interface modal matrix and $\Psi_{i 3}$ is the interior partition of the constraint-mode matrix.

An orthogonal projection technique for model-order reduction seeks an approximation of $x(t)$ constrained to stay in the subspace spanned by the columns of $V_{n}$, namely

$$
x(t) \approx V_{n} z(t)
$$

Then by imposing the so-called Galerkin orthogonal condition:

$$
M V_{n} \ddot{z}(t)+K Q_{n} z(t)-B u(t) \perp \operatorname{span}\left\{V_{n}\right\} .
$$

it yields a reduced-order system:

$$
\Sigma_{n}:\left\{\begin{array}{rl}
M_{n} \ddot{z}(t)+K_{n} z(t) & =B_{n} u(t)  \tag{2.9}\\
\hat{y}(t) & =L_{n}^{T} z(t)
\end{array},\right.
$$

where $M_{n}=V_{n}^{T} M V_{n}, K_{n}=V_{n}^{T} K V_{n}, B_{n}=V_{n}^{T} B$ and $L_{n}=V_{n}^{T} L$. By the definition of $V_{n}$, the matrices $M_{n}$ and $K_{n}$ of the reduced system $\Sigma_{n}$ are of the following forms

$$
M_{n}=\left[\begin{array}{ccc}
I & & M_{13}^{(n)} \\
& I & M_{23}^{(n)} \\
\left(M_{13}^{(n)}\right)^{T} & \left(M_{13}^{(n)}\right)^{T} & \widehat{M}_{33}
\end{array}\right] \text { and } K_{n}=\left[\begin{array}{ccc}
\Lambda_{1}^{(n)} & & \\
& \Lambda_{2}^{(n)} & \\
& & \widehat{K}_{33}
\end{array}\right]
$$

where

$$
\begin{aligned}
M_{i 3}^{(n)} & =\Phi_{i}^{T} \widehat{M}_{i 3} \quad \text { and } \quad \widehat{M}_{i 3}=M_{i 3}-M_{i i} K_{i i}^{-1} K_{i 3} \quad \text { for } i=1,2, \\
\widehat{M}_{33} & =M_{33}-\sum_{i=1}^{2}\left(K_{i 3}^{T} K_{i i}^{-1} M_{i 3}+M_{i 3}^{T} K_{i i}^{-1} K_{i 3}-K_{i 3}^{T} K_{i i}^{-1} M_{i i} K_{i i}^{-1} K_{i 3}\right),
\end{aligned}
$$

and $\widehat{K}_{33}$ is the Schur complement of $\operatorname{diag}\left(K_{11}, K_{22}\right)$ in $K$ of the form

$$
\widehat{K}_{33}=K_{33}-K_{13}^{T} K_{11}^{-1} K_{13}-K_{23}^{T} K_{22}^{-1} K_{23}
$$

and $\Lambda_{i}^{(n)}=\operatorname{diag}\left(\lambda_{1}^{(i)}, \lambda_{2}^{(i)}, \ldots, \lambda_{n_{i}}^{(i)}\right)$.
A high-level description of a generic CMS method is as followings.

## Generic CMS Method

1. Compute the selected eigenpairs $\left(\lambda_{j}^{(i)}, \boldsymbol{\phi}_{j}^{(i)}\right)$ of the generalized eigenproblems $K_{i i} \phi_{j}^{(i)}=\lambda_{j}^{(i)} M_{i i} \phi_{j}^{(i)}$ for $i=1,2$,
2. Retain some eigenpairs $\left(\lambda_{j}^{(i)}, \phi_{j}^{(i)}\right)$ to define transformation matrix $V_{n}$,
3. Form $M_{n}, K_{n}, B_{n}, L_{n}$ to define the reduced system $\Sigma_{n}$ as in (2.9).

In the standard CMS method, the $n_{i}$ modes $\phi_{j}^{(i)}$ associated with smallest eigenvalues $\lambda_{j}^{(i)}$ are retained to define the projection matrix $V_{n} . V_{n}$ is called the Craig-Bampton transformation matrix in structure dynamics [3].

In an alternative method, which we call the $\mathrm{CMS}_{\chi}$, the $n_{i}$ modes $\phi_{j}^{(i)}$ in $V_{n}$ are selected according to the highest norm of the rank-one coupling matrices $S_{j}^{(i)}$ :

$$
\begin{equation*}
S_{j}^{(i)}=\frac{1}{\lambda_{j}^{(i)}} \widehat{M}_{i 3}^{T} \phi_{j}^{(i)}\left(\phi_{j}^{(i)}\right)^{T} \widehat{M}_{i 3} \tag{2.10}
\end{equation*}
$$

Therefore, the selected modes $\phi_{j}^{(i)}$ in $\mathrm{CMS}_{\chi}$ may not be in the natural order as in CMS. As a result, to find such $n_{i}$ modes, we may have to find more than $n_{i}$ smallest eigenpairs of the matrix pairs $\left(M_{i i}, K_{i i}\right)$. This will be shown by numerical examples in section 4. But first we give a justification for the $\mathrm{CMS}_{\chi}$ method in the next section.

## 3 Derivation of $\mathrm{CMS}_{\chi}$

Let us assume that $\Phi_{i}$ contains all $N_{i}$ modes of the submatrix pairs $\left(M_{i i}, K_{i i}\right)$ for $i=1,2$. Then the system $\Sigma_{N}$ in its modal coordinate in frequency domain is of the form

$$
\left(-\omega^{2}\left[\begin{array}{ccc}
I & & M_{13}^{(N)}  \tag{3.11}\\
& I & M_{23}^{(N)} \\
\left(M_{13}^{(N)}\right)^{T} & \left(M_{23}^{(N)}\right)^{T} & \widehat{M}_{33}
\end{array}\right]+\left[\begin{array}{ccc}
\Lambda_{1}^{(N)} & & \\
& \Lambda_{2}^{(N)} & \\
& & \widehat{K}_{33}
\end{array}\right]\right)\left[\begin{array}{l}
X_{1}(\omega) \\
X_{2}(\omega) \\
X_{3}(\omega)
\end{array}\right]=\left[\begin{array}{c}
B_{1}^{(N)} \\
B_{2}^{(N)} \\
\widehat{B}_{3}
\end{array}\right] U(\omega) .
$$

For the sake of notation, we will drop the superscript ${ }^{(N)}$ in the rest of section. By solving $X_{1}(\omega)$ and $X_{2}(\omega)$ from the first and second equations of 3.11) and then substituting into the third interface equation of the (3.11), it yields

$$
\begin{align*}
\left(\omega ^ { 4 } \sum _ { i = 1 } ^ { 2 } \left[-M_{i 3}^{T}(-\omega I+\right.\right. & \left.\left.\left.\Lambda_{i}\right)^{-1} M_{i 3}\right]-\omega^{2} \widehat{M}_{33}+\widehat{K}_{33}\right) X_{3}(\omega) \\
& =\left(\omega^{2} \sum_{i=1}^{2}\left[M_{i 3}^{T}\left(-\omega I+\Lambda_{i}\right)^{-1} B_{i}\right]+\widehat{B}_{3}\right) U(\omega) \tag{3.12}
\end{align*}
$$

In the context of structural dynamics, the equation (3.12) presents the force applied to the interface and applied to it by the subsystems.

Instead of solving equation (3.12) for $X_{3}(\omega)$ directly, we simplify the equation first, since we are only interested in looking for "important modes". An approximation of (3.12) is taking the first three terms of the power expansion in $\omega^{2}$ of the coefficient matrix on the left hand side, and taking the constant term on the right hand side. This yields an approximate equation of (3.12):

$$
\begin{equation*}
\left[-\omega^{4}\left(M_{13}^{T} \Lambda_{1}^{-1} M_{13}+M_{23}^{T} \Lambda_{2}^{-1} M_{23}\right)-\omega^{2} \widehat{M}_{33}+\widehat{K}_{33}\right] \widetilde{X}_{3}(\omega)=\widehat{B}_{3} U(\omega) \tag{3.13}
\end{equation*}
$$

Let the power series expansion of $\widetilde{X}_{3}(\omega)$ be formally denoted by

$$
\tilde{X}_{3}(\omega)=\left(\sum_{\ell=0}^{\infty} r_{\ell} \omega^{2 \ell}\right) U(\omega),
$$

where $r_{\ell}$ are called the $\ell$-th moment (vector) of $\widetilde{X}_{3}(\omega)$. Then by comparing the two sides of equation (3.13) in the power of $\omega^{2}$, the moments $r_{\ell}$ are given by

$$
\begin{aligned}
r_{0} & =\widehat{K}_{33}^{-1} \widehat{B}_{3}, \\
r_{1} & =\widehat{K}_{33}^{-1} \widehat{M}_{33} r_{0}, \\
r_{\ell} & =\widehat{K}_{33}^{-1}\left(\widehat{M}_{33} r_{\ell-1}+\left(\sum_{i=1}^{2} M_{i 3}^{T} \Lambda_{i}^{-1} M_{i 3}\right) r_{\ell-2}\right) \quad \text { for } \ell \geq 2 .
\end{aligned}
$$

By an exactly analogous calculation, for the reduced-order system $\Sigma_{n}$ in its modal coordinates form, namely

$$
M_{n}=\left[\begin{array}{ccc}
I & & M_{13}^{(n)} \\
& I & M_{23}^{(n)} \\
\left(M_{13}^{(n)}\right)^{T} & \left(M_{13}^{(n)}\right)^{T} & M_{33}^{(n)}
\end{array}\right], K_{n}=\left[\begin{array}{lll}
\Lambda_{1}^{(n)} & & \\
& \Lambda_{2}^{(n)} & \\
& & K_{33}^{(n)}
\end{array}\right]
$$

and

$$
B_{n}=\left[\begin{array}{l}
B_{1}^{(n)} \\
B_{2}^{(n)} \\
B_{33}^{(n)}
\end{array}\right], L_{n}=\left[\begin{array}{l}
L_{1}^{(n)} \\
L_{2}^{(n)} \\
L_{33}^{(n)}
\end{array}\right] .
$$

The moment vectors $r_{\ell}^{(n)}$ for the solution $\widetilde{X}_{3}^{(n)}(\omega)$ of the approximate interface equation are given by

$$
\begin{aligned}
r_{0}^{(n)} & =\left(K_{33}^{(n)}\right)^{-1} B_{3}^{(n)}, \\
r_{1}^{(n)} & =\left(K_{33}^{(n)}\right)^{-1} M_{33}^{(n)} r_{0}^{(n)}, \\
r_{\ell}^{(n)} & =\left(K_{33}^{(n)}\right)^{-1}\left(M_{33}^{(n)} r_{\ell-1}^{(n)}+\left(\sum_{i=1}^{2}\left(M_{i 3}^{(n)}\right)^{T}\left(\Lambda_{i}^{(n)}\right)^{-1} M_{i 3}^{(n)}\right) r_{\ell-2}^{(n)}\right) \quad \text { for } \ell \geq 2 .
\end{aligned}
$$

Note that the dimensions of the moment vectors $\left\{r_{\ell}\right\}$ and $\left\{r_{\ell}^{(n)}\right\}$ are the same since we assume that the DOFs of the interface block is unchanged.

A natural optimal strategy to define a reduced-order system $\Sigma_{n}$ is to match or approximate as many moments as possible. To match the first moment $r_{0}=r_{0}^{(n)}$, it suggests that

$$
K_{33}^{(n)}=\widehat{K}_{33} \quad \text { and } \quad B_{3}^{(n)}=\widehat{B}_{3} .
$$

To match the second moment $r_{1}=r_{1}^{(n)}$, it derives that

$$
M_{33}^{(n)}=\widehat{M}_{33} .
$$

Unfortunately, there is no easy way to match the third moment $r_{2}$ exactly. Instead, we try to minimize the difference between $r_{2}$ and $r_{2}^{(n)}$ :

$$
\begin{align*}
\left\|r_{2}-r_{2}^{(n)}\right\|_{2} & =\left\|\widehat{K}_{33}^{-1}\left(\sum_{i=1}^{2} M_{i 3}^{T} \Lambda_{i}^{-1} M_{i 3}-\left(M_{i 3}^{(n)}\right)^{T}\left(\Lambda_{i}^{(n)}\right)^{-1} M_{i 3}^{(n)}\right) \widehat{K}_{33}^{-1} \widehat{B}_{3}\right\|_{2} \\
& \leq c\|\underbrace{\sum_{j=1}^{N_{1}} S_{j}^{(1)}-\sum_{j=1}^{n_{1}}\left(S_{j}^{(1)}\right)^{(n)}}_{1}+\underbrace{\sum_{j=1}^{N_{2}} S_{j}^{(2)}-\sum_{j=1}^{n_{2}}\left(S_{j}^{(2)}\right)^{(n)}}_{2}\|_{2} \tag{3.14}
\end{align*}
$$

where $c=\left\|\widehat{K}_{33}^{-1}\right\|_{2}\left\|\widehat{K}_{33}^{-1} \widehat{B}_{3}\right\|_{2}$, a constant independent of the modes $\phi_{j}^{(i)} . S_{j}^{(i)}$ and $\left(S_{j}^{(i)}\right)^{(n)}$ are the coupling matrices for the $j$-th mode of the subsystem $i$ as defined in (2.10). Assume that $S_{j}^{(i)}$ and $\left(S_{j}^{(i)}\right)^{(n)}$ are in descending order according to their norms, respectively,

$$
\left\|S_{1}^{(i)}\right\| \geq\left\|S_{2}^{(i)}\right\| \geq \cdots \geq\left\|S_{N_{i}}^{(i)}\right\|, \quad\left\|\left(S_{1}^{(i)}\right)^{(n)}\right\| \geq\left\|\left(S_{2}^{(i)}\right)^{(n)}\right\| \geq \cdots \geq\left\|\left(S_{n_{i}}^{(i)}\right)^{(n)}\right\|
$$

The best we can do is to set

$$
\left(S_{j}^{(i)}\right)^{(n)}=S_{j}^{(i)} \quad \text { for } \quad j=1,2, \ldots, n_{i}
$$

This cancels out the first $n_{i}$ terms of the differences labled as 1 and 2 of the upper bound in (3.14), and leaves the sums of the remaining terms smallest possible. This yields the $\mathrm{CMS}_{\chi}$-mode selection rule as we described in section 2: retain the first $n_{i}$ modes of the subsystem $i$ according to the largest norms of the coupling matrices $S_{j}^{(i)}$.

Note that the matrices $\widehat{M}_{i 3}$ which couples subsystems and the interface are included in the coupling matrices $S_{j}^{(i)}$. Therefore, they are reflected for the retention of modes of importance. These coupling effects are essentially ignored by the CMS mode selection. To this end, we also note that $\mathrm{CMS}_{\chi}$-mode selection criterion is essentially the same as the one in the OMR method derived by the DtN mapping [16], but without the assumption of the special form of the external force term $B u(t)$ in the original system $\Sigma_{N}$ (1.1).

## 4 Numerical Experiments

In this section, we present two numerical examples to compare the two mode selection criteria discussed in this paper. All numerical experiments were run in MATLAB on a Linux Server with Dual 1.2Ghz CPUs and 2GB of memory.


Fig. 2. Left: magnitudes (in $\log$ of base 10) of the transfer functions (top) and relative errors (bottom). Right: retained modes of subsystems by CMS and $\mathrm{CMS}_{\chi}$

Example 1. In this example, the mass and stiffness matrices $M$ and $K$ are from HarwellBoeing BCS sparse matrix collection [5]. The number of DOFs of $\Sigma_{N}$ is $N=420$, and that of two subsystems are $N_{1}=190$ and $N_{2}=194$, respectively. The top left plot of Fig. 2] shows the magnitude (in $\log$ of base 10) of the transfer function $H(\omega)$ of the SISO system $\Sigma_{N}$ with $B=L=\left[\begin{array}{llll}1 & 0 & \ldots & 0\end{array}\right]^{T}$. The transfer functions $H_{n}^{\text {CMS }}(\omega)$ and $H_{n}^{\mathrm{CMS}_{\chi}}(\omega)$ of the reduced systems $\Sigma_{n}$, computed by CMS and $\mathrm{CMS}_{\chi}$, are shown in the same plot. The number of DOFs of reduced-order systems $\Sigma_{n}$ is $n=153$ with $n_{1}=52$ and $n_{2}=65$, respectively. The relative errors $\left|H(\omega)-H_{n}^{\mathrm{CMS}}(\omega)\right| /|H(\omega)|$ and $\left|H(\omega)-H_{n}^{\mathrm{CMS}_{\chi}}(\omega)\right| /|H(\omega)|$ shown in the lower left plot of Fig. 2 indicate that $H_{n}^{\mathrm{CMS}_{\chi}}(\omega)$ is a much accurate approximation of $H(\omega)$ than $H_{n}^{\mathrm{CMS}}(\omega)$, under the same order of reduced DOFs.

Two right plots of Fig. 2 show the eigenvalues of original systems and the ones retained by CMS and $\mathrm{CMS}_{\chi}$. Note again that the numbers of eigenvalues of subsystems retained by the two methods are the same. $\mathrm{CMS}_{\chi}$ skips some of lower frequency eigenvalues, and uses some higher frequency eigenvalues to take into the account of coupling effects between the subsystems and the interface. On the other, CMS simply takes the lowest frequency eigenvalues in order.

Example 2. This is a SISO system $\Sigma_{N}$ arised from simulation of a prototype radiofrequency MEMS filter [2]. The DOFs of $\Sigma_{N}$ is $N=490$ and that of two subsystems are $N_{1}=N_{2}=238$. The DOFs of interface is $N_{3}=14$. Fig. 3 shows the transfer functions $H(\omega), H_{n}^{\mathrm{CMS}}(\omega)$ and $H_{n}^{\mathrm{CMS}_{\chi}}(\omega)$. The DOFs of reduced subsystems by the
both methods are $n_{1}=n_{2}=85$. The relative errors $\left|H(\omega)-H_{n}^{\mathrm{CMS}}(\omega)\right| /|H(\omega)|$ and $\left|H(\omega)-H_{n}^{\mathrm{CMS}_{\chi}}(\omega)\right| /|H(\omega)|$ in the lower left plot of Fig. 3 show the improvement made by the new $\mathrm{CMS}_{\chi}$ method. Two right plots of Fig. 3 show the differences in the retention of the same number of modes of subsystems.


Fig. 3. Left: magnitudes (in $\log$ of base 10) of the transfer functions (top) and relative errors (bottom). Right: retained modes of subsystems by CMS and $\mathrm{CMS}_{\chi}$

## 5 Conclusion Remarks

A new coupling matrix-based mode selection criterion for the popular CMS method is presented in this paper. It is derived based on moment-matching property for the interface solution. Our work is motivated by the recent work of Givoli et al [16], in which the term "coupling matrix" is coined. Our mode selection criterion is compatible to the one proposed by Givoli et al, which uses Dirichlet-to-Neumann (DtN) mapping as an analysis tool. The performance improvement of the new mode selection criterion is demonstrated by numerical examples.

The coupling matrix-based mode selection costs more than the standard one, since some extra eigenpairs of the subsystems are typically required. If the sizes of subsystems are moderate, the extra cost may not be significant measured by the CPU time. Multilevel substructuring with an optimal mode selection is a subject of future study. It is worth to note that modal reduction methods as discussed in this paper are generally less accurate and efficient than Krylov subspace-based reduction methods. A Krylov subspace-based substructuring method is in progress.

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## References

1. P.E. Barbone and D. Givoli. Optimal modal reduction of vibrating substructures. Int. J. Numer. Meth. Engng, 57:341-369, 2003.
2. D. Bindel, Z. Bai, and J. Demmel. Reduced order models in Microsystems and RF MEMS. To appear in the proceedings of PARA'04: Workshop on the state-of-the-art in scientific computing, Lyngby, Denmark, June 20-23, 2004.
3. R. R. Craig Jr. Coupling of substructures for dynamic analysis - an overview. AIAA-2000-1573, 2000.
4. R. R. Craig Jr. and M. C. C. Bampton. Coupling of substructures for dynamic analysis. AIAA Journal, 6(7):1313-1319, 1968.
5. I.S. Duff, R.G. Grimes, and J.G. Lewis. Users' guide for the Harwell-Boeing sparse matrix collection (release 1). Technical Report RAL-92-086, Rutherford Appleton Laboratory, December 1992. Available at the MatrixMarket: http://math.nist.gov/MatrixMarket.
6. D. Givoli, P. E. Barbone, and I. Patlashenko. Which are the important modes of a subsystem? Int. J. Numer. Meth. Engng, 59:1657-1678, 2004.
7. W. C. Hurty. Vibrations of structural systems by component-mode synthesis. Journal of the Engineering Mechanics Division, ASCE, 86:59-69, 1960.
8. W. C. Hurty. Dynamic analysis of structural systems using component modes. AIAA Journal, 3:678-685, 1965.
9. J.B. Keller and D. Givoli. Exact non-reflecting boundary conditions. Journal of Computational Physics, 82:172-192, 1989.
